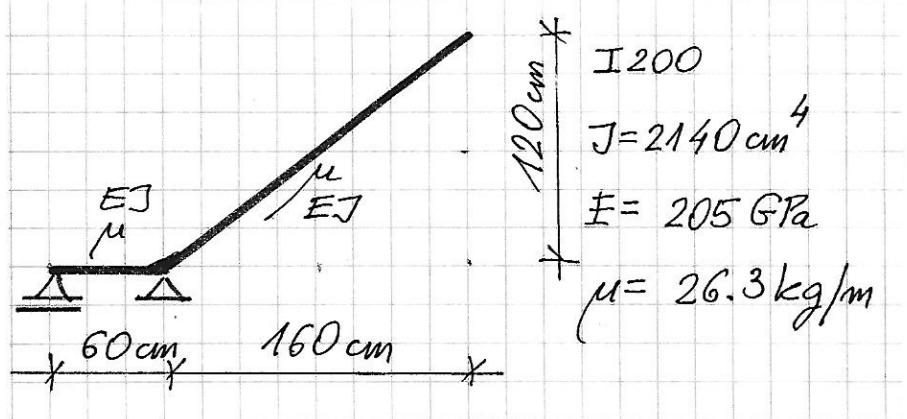


Egzamin pisemny z Mechaniki Konstrukcji II, 8 IX 2016 r.

NAZWISKO imię				
Grupa	Data zaliczenia ćwiczeń		Numer albumu	
Ocena zadania 1	Ocena zadania 2	Ocena zadania 3	Ocena z egzaminu	Ocena łączna
				Data

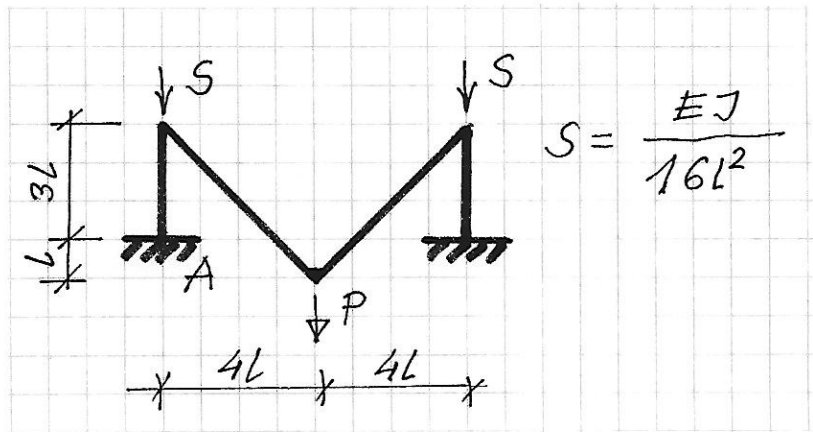
Zadanie 1

Znaleźć pierwszą częstość drgań własnych danej ramy.
(Compute the first eigenfrequency of the given frame)



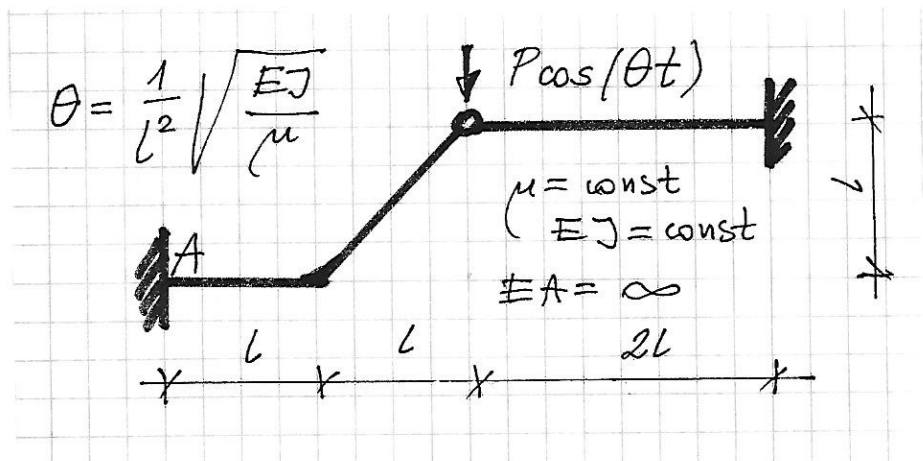
Zadanie 2

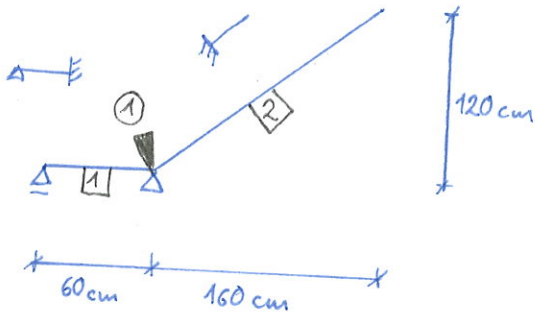
Rozważamy ramę poddaną dużym siłom osiowym S i zginaną siłą P . Znaleźć moment zginający w utwierdzeniu A .
(Consider the frame subject to a big axial force S and bent by the force P . Find the bending moment at the clamped edge A).



Zadanie 3

Dana jest rama obciążona siłą harmonicznie zmienną w czasie, por. rys. Znaleźć amplitudę M_A .
(The given frame is subject to a harmonic load, see fig. Compute the amplitude of M_A .)





$$J = 2140 \text{ cm}^4 = 2,14 \cdot 10^{-5} \text{ m}^4$$

$$E = 205 \text{ GPa} = 205 \cdot 10^6 \frac{\text{kN}}{\text{m}^2}$$

$$\mu = 26,3 \frac{\text{kg}}{\text{m}} \approx 26,3 \frac{\text{N}}{\text{m}} \cdot \frac{\text{s}^2}{\text{m}} = 0,0263 \frac{\text{kN s}^2}{\text{m}^2}$$

$$\left[\text{N} = \text{kg} \cdot \frac{\text{m}}{\text{s}^2} \right]$$

Przyjmijmy $l=1\text{m}$. Let $l=1\text{m}$

$$\lambda = l \sqrt[4]{\frac{\mu \omega^2}{EJ}}$$

$$l_1 = 0,6l$$

$$\lambda_1 = 0,6l \sqrt[4]{\frac{\mu \omega^2}{EJ}} = 0,6\lambda$$

$$l_2 = 2l$$

$$\lambda_2 = 2l \sqrt[4]{\frac{\mu \omega^2}{EJ}} = 2\lambda$$

$$\eta = [\varphi_1]$$

$$\Phi_1^1 + \Phi_1^2 = 0$$

$$\Phi_1^1 = \frac{EJ}{l_1} [\alpha'(\lambda_1) \varphi_1] = \frac{EJ}{0,6l} \alpha'(0,6\lambda) \varphi_1 = \frac{5}{3} \frac{EJ}{l} \alpha'(0,6\lambda) \varphi_1$$

$$\Phi_1^2 = \frac{EJ}{l_2} [\alpha''(\lambda_2) \varphi_1] = \frac{EJ}{2l} \alpha''(2\lambda) \varphi_1$$

$$\frac{EJ}{l} \left[\frac{5}{3} \alpha'(0,6\lambda) + \frac{1}{2} \alpha''(2\lambda) \right] \varphi_1 = 0$$

Szukamy rozwiązania nietrywialnego.
We look for the non-trivial solution

$$\varphi_1 \neq 0$$

$$\frac{5}{3} \alpha'(0,6\lambda) + \frac{1}{2} \alpha''(2\lambda) = 0$$

$$\alpha'(0,6\lambda) + \frac{3}{10} \alpha''(2\lambda) = 0 \Rightarrow \alpha'(0,6\lambda) = -\frac{3}{10} \alpha''(2\lambda)$$

Korzystamy z tablic. We use the tables.

$$\lambda = 0,8: \quad \alpha'(0,6 \cdot 0,8) \approx \alpha'(0,5) = 2,999$$

$$-\frac{3}{10} \alpha''(2 \cdot 0,8) = -\frac{3}{10} \alpha''(1,6) = 1,373$$

$$\lambda = 0,9: \quad \alpha'(0,6 \cdot 0,9) \approx \alpha'(0,7) = 2,995$$

$$-\frac{3}{10} \alpha''(2 \cdot 0,9) = -\frac{3}{10} \alpha''(1,8) = 6,787$$

Rozwiązanie.
Solution.

$$\lambda \in (0,8; 0,9)$$

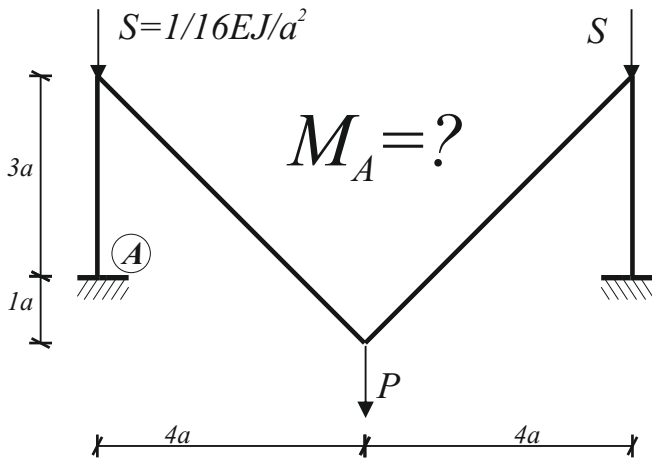
Przyjmijmy
Let

$$\lambda = 0,85$$

$$\lambda = l \sqrt[4]{\frac{\mu \omega^2}{EJ}} \Rightarrow \omega = \frac{\lambda^2}{l^2} \sqrt{\frac{EJ}{\mu}} = \left(\frac{0,85}{1,0 \text{ m}} \right)^2 \sqrt{\frac{205 \cdot 10^6 \frac{\text{kN}}{\text{m}^2} \cdot 2,14 \cdot 10^{-5} \text{ m}^4}{0,0263 \frac{\text{kN s}^2}{\text{m}^2}}}$$

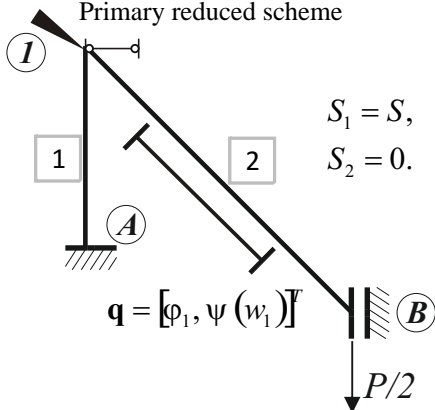
$$\omega \approx 295,1 \text{ [Hz]}$$

Zadanie 2 Problem 2



$M_A = ?$

Zastępczy schemat zredukowany
Primary reduced scheme



$S_1 = S,$
 $S_2 = 0.$

Plan przesunięć
Translation plan

$\psi_1 = \psi,$
 $\psi_2 = \frac{3}{4}\psi,$
 $\bar{L}_z = \frac{P}{2} 4a \left(\frac{3}{4} \bar{\psi} \right) = \frac{3}{2} Pa \bar{\psi},$
 $\bar{L}_s = S 3a \psi \bar{\psi} = \frac{3}{16} \frac{EJ}{a} \psi \bar{\psi}.$

Równania równowagi. Equilibrium equations

1. $\Phi_1^1 + \Phi_1^2 = 0$

2. $\{\bar{\psi} = -1\} [\Phi_A^1 + \Phi_1^1] \{-1\} + [\Phi_1^2 + \Phi_B^2] \left\{ -\frac{3}{4} \right\} - \frac{3}{16} \frac{EJ}{a} \psi - \frac{3}{2} Pa = 0$

Wzory transformacyjne.

Slope-deflection equations.

$\sigma_1 = 3a \sqrt{\frac{S}{EJ}} = 0.75,$

$\sigma_2 = 0.$

	$\frac{EJ}{a} \varphi_1$	$\frac{EJ}{a} \psi$			
F11	1.30815	-1.9812	+	=	-0.2348
FA1	0.67303	-1.9812	+	=	-1.3838
F12	0.70711	-0.7955	+	=	0.2348
FB2	0.35355	-0.7955	+	=	-0.4049

Pa

Układ równań i rozwiązanie.

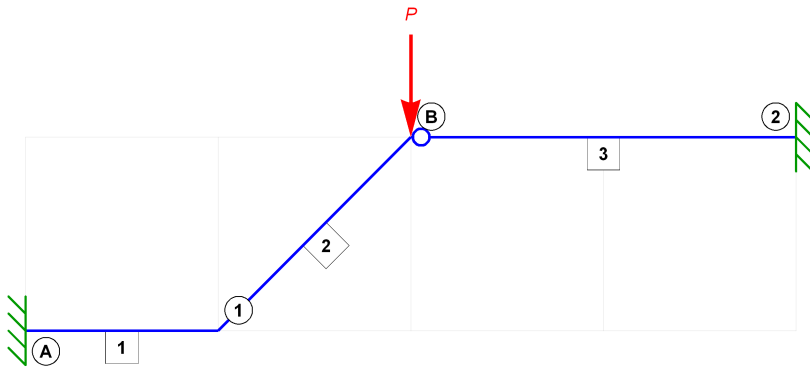
System of equations and its solution.

$\frac{EJ}{a} \begin{bmatrix} 2.0153E+00 & -2.7767E+00 \\ -2.7767E+00 & 4.9681E+00 \end{bmatrix} \mathbf{q} = \begin{bmatrix} 1 \\ 1/2 \end{bmatrix} Pa$

Rozwiązanie. Solution.

$\mathbf{q} = \begin{bmatrix} 1.809E+00 \\ 1.313E+00 \end{bmatrix} \frac{Pa^2}{EJ}$

Zadanie 3. Problem 3.



$$\lambda_1 = 1$$

$$\lambda_2 = \sqrt{2}$$

$$\lambda_3 = 2$$

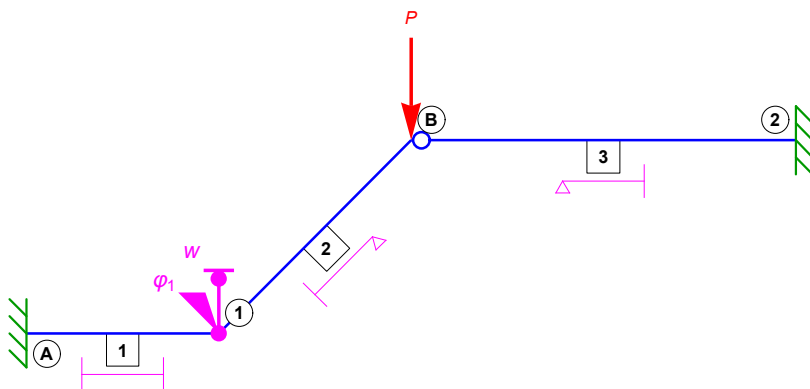
Wektor niewiadomych.

Vector of unknowns.

$$\mathbf{q} = \begin{pmatrix} \varphi_1 \\ \frac{w}{l} \\ 1 \end{pmatrix}$$

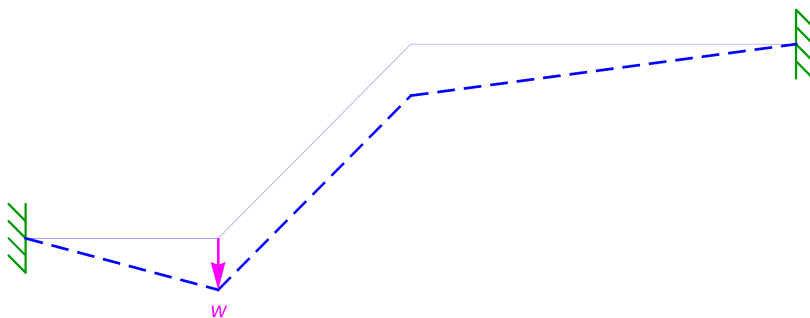
Układ zastępczy.

The primary structure.



Plan przesunięć.

Translation plan.



	$w_{\frac{1}{2}}^K$	$w_{\frac{2}{2}}^K$	u^K
Pręt 1 :	$w_{\frac{1}{2}}^1 = 0$	$w_{\frac{1}{2}}^1 = w$	$u^1 = 0$
Pręt 2 :	$w_{\frac{1}{2}}^2 = \frac{1}{\sqrt{2}} w$	$w_{\frac{2}{2}}^2 = \frac{1}{\sqrt{2}} w$	$u^2 = -\frac{1}{\sqrt{2}} w$
Pręt 3 :	$w_{\frac{3}{2}}^3 = w$	$w_{\frac{2}{2}}^3 = 0$	$u^3 = 0$

Wyjściowe siły brzegowe.

Initial end forces.

$$W_{\frac{3}{2}}^0 = -P$$

Wzory transformacyjne.

Slope-deflection equations.

$$\Phi_{\frac{1}{2}}^1 = \frac{EJ}{1} [\alpha(1) \varphi_1 - \vartheta(1) \frac{w}{1}] = \frac{EJ}{1} [3.990 \varphi_1 - 5.948 \frac{w}{1}]$$

$$\Phi_{\frac{1}{2}}^2 = \frac{EJ}{1} [\frac{1}{\sqrt{2}} \alpha'(\sqrt{2}) \varphi_1 + \{ \frac{1}{2\sqrt{2}} \vartheta'(\sqrt{2}) - \frac{1}{2\sqrt{2}} \delta'(\sqrt{2}) \} \frac{w}{1}] = \frac{EJ}{1} [2.067 \varphi_1 - 0.179 \frac{w}{1}]$$

$$W_{\frac{1}{2}}^1 = \frac{EJ}{1^2} [-\vartheta(1) \varphi_1 + \gamma(1) \frac{w}{1}] = \frac{EJ}{1^2} [-5.948 \varphi_1 + 11.628 \frac{w}{1}]$$

$$W_{\frac{1}{2}}^2 = \frac{EJ}{1^2} [\frac{1}{2} \vartheta'(\sqrt{2}) \varphi_1 + \{ \frac{1}{4} \gamma'(\sqrt{2}) - \frac{1}{4} \varepsilon'(\sqrt{2}) \} \frac{w}{1}] = \frac{EJ}{1^2} [1.326 \varphi_1 - 0.633 \frac{w}{1}]$$

$$W_{\frac{2}{2}}^2 = \frac{EJ}{1^2} [-\frac{1}{2} \delta'(\sqrt{2}) \varphi_1 + \{ -\frac{1}{4} \varepsilon'(\sqrt{2}) + \frac{1}{4} \chi'(\sqrt{2}) \} \frac{w}{1}] = \frac{EJ}{1^2} [-1.580 \varphi_1 - 0.380 \frac{w}{1}]$$

$$W_{\frac{3}{2}}^3 = \frac{EJ}{1^2} [\frac{1}{8} \chi'(2) \frac{w}{1}] - P = \frac{EJ}{1^2} [-0.115 \frac{w}{1}] - P$$

Osiowe siły bezwładności.

Axial inertia forces.

$$B_{\frac{1}{2}}^1 = \vartheta^2 \cdot \mu \cdot \sqrt{2} \cdot 1 \cdot (-\frac{1}{\sqrt{2}} w) = \frac{EJ}{1^2} [-\frac{w}{1}]$$

Równania równowagi.

Equations of equilibrium.

$$\Phi_{\frac{1}{2}}^1 + \Phi_{\frac{1}{2}}^2 = 0$$

$$-W_{\frac{1}{2}}^1 \cdot \bar{w} - W_{\frac{1}{2}}^2 \cdot \frac{1}{\sqrt{2}} \bar{w} - W_{\frac{2}{2}}^2 \cdot \frac{1}{\sqrt{2}} \bar{w} - W_{\frac{3}{2}}^3 \cdot \bar{w} + B_{\frac{1}{2}}^1 \cdot (-(1/\sqrt{2}) \bar{w}) = \bar{0}$$

$$\frac{EJ}{1} \begin{pmatrix} 6.057 & -6.127 \\ -6.127 & 10.090 \end{pmatrix} \begin{pmatrix} \varphi_1 \\ \frac{w}{1} \end{pmatrix} = 1 P \begin{pmatrix} 0 \\ 1.000 \end{pmatrix}$$

Rozwiązanie.

Solution.

$$\mathbf{q} = \begin{pmatrix} \varphi_1 \\ \frac{w}{1} \end{pmatrix} = \frac{1^2 P}{EJ} \begin{pmatrix} 0.260 \\ 0.257 \end{pmatrix}$$

Siły i momenty brzegowe.

End forces and moments.

$$\Phi_{\frac{1}{2}}^1 = -1.028 \text{ l P}$$

$$\Phi_{\frac{1}{2}}^2 = -0.491 \text{ l P}$$

$$\Phi_{\frac{1}{2}}^3 = 0.491 \text{ l P}$$

$$\Phi_{\frac{2}{2}}^2 = 0.236 \text{ l P}$$

$$W_{\frac{1}{2}}^1 = -1.549 \text{ P}$$

$$W_{\frac{1}{2}}^2 = 1.442 \text{ P}$$

$$W_{\frac{1}{2}}^3 = 0.182 \text{ P}$$

$$W_{\frac{2}{2}}^2 = -0.508 \text{ P}$$

$$W_{\frac{3}{2}}^3 = -1.029 \text{ P}$$

$$W_{\frac{2}{2}}^3 = -0.174 \text{ P}$$