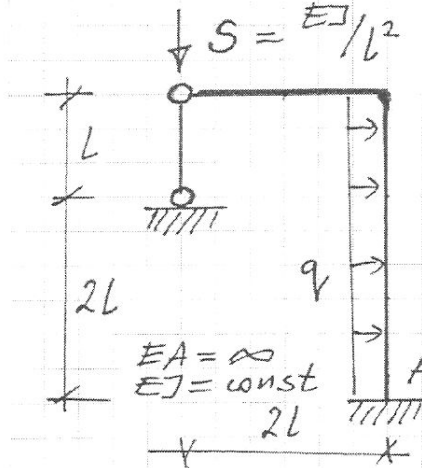


NAZWISKO Imię:				
ocena zadania 1	ocena zadania 2	ocena zadania 3	ocena egz. pis.	Ocena Ostateczna
				Ocena łączna
				Data

Problem 1

Dana jest rama obciążona dużą siłą osiową i poddana obciążeniu q . Znaleźć moment w utwierdzeniu A.

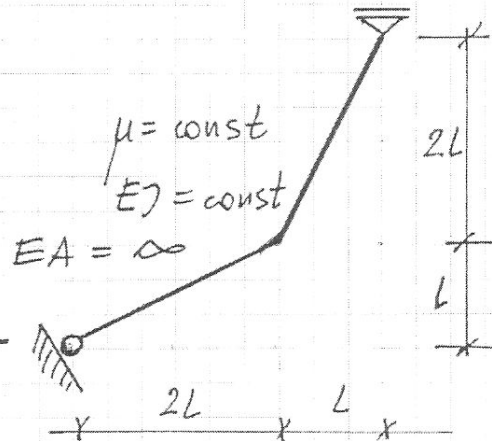
(Find the bending moment at the clamped node A of the given plane frame subjected to the lateral load of intensity q and subjected to the given big axial force.)



Problem 2

Dana jest rama nieważka z masą skupioną, poddana obciążeniu harmonicznemu. Znaleźć amplitudę przemieszczenia masy.

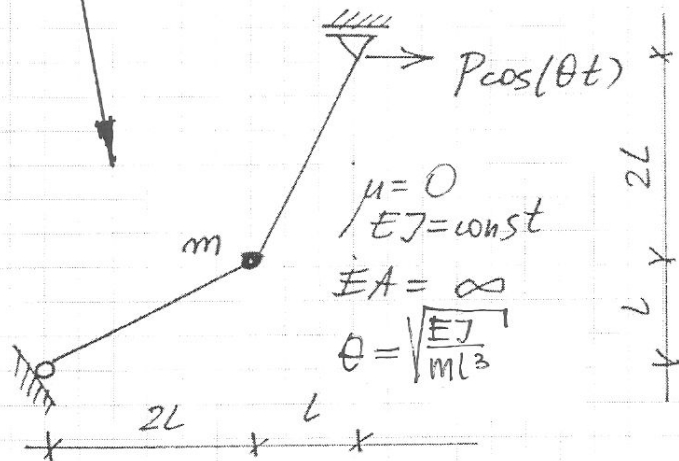
(The given frame of weightless bars, with one concentrated mass, is subject to the harmonic load. Find the amplitude of the displacement of the mass.)



Problem 3

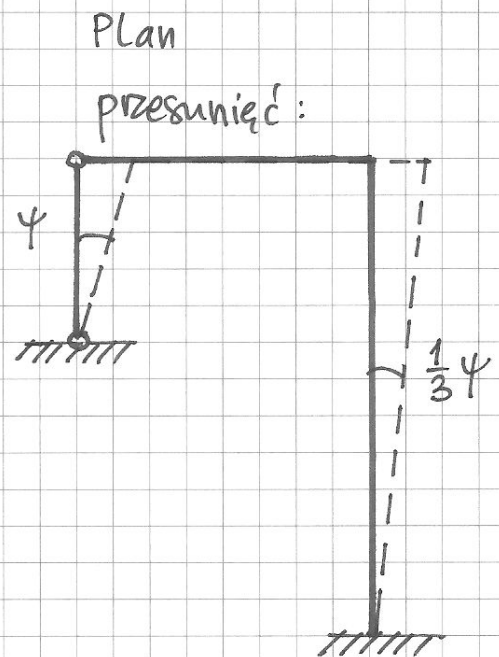
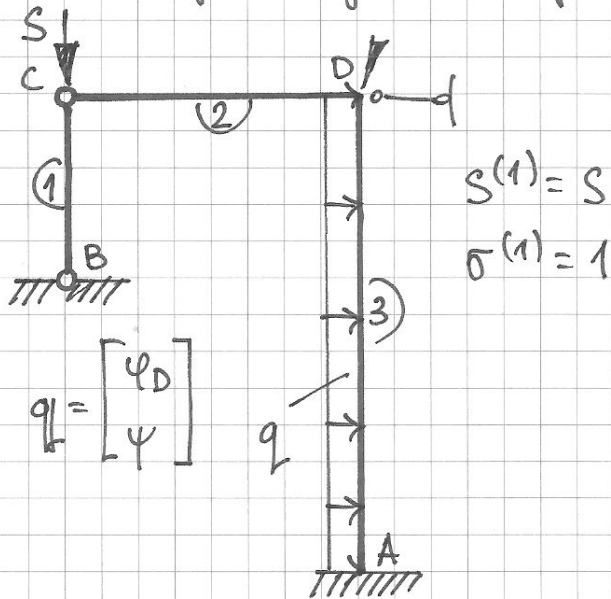
Zapisać równania określające częstości drgań własnych danej ramy płaskiej o ciągłym rozkładzie masy.

(Find the equations which determine the eigenfrequencies of the given plane frame of a continuous mass distribution.)



Egzamin z MK2, 11 IX 2014, zadanie 1

Schemat geometrycznie wyznaczalny



Równania równowagi

$$\bar{\Phi}_D^{(2)} + \bar{\Phi}_D^{(3)} = 0$$

$$[\bar{\Phi}_A^{(3)} + \bar{\Phi}_D^{(3)}] \cdot \frac{1}{3}\bar{\psi} + S \cdot L \cdot \psi \cdot \bar{\psi} + q \cdot 3L \cdot \frac{3}{2}L \cdot \frac{1}{3}\bar{\psi} = 0$$

Wzory transformacyjne i siły wewnętrzne wyjściowe

$$\bar{\Phi}_D^{(2)} = \frac{3EJ}{2L} [\psi_D]$$

$$\bar{\Phi}_A^{(3)} = \frac{2EJ}{3L} \left[\psi_D - 3 \cdot \frac{1}{3}\psi \right] - \frac{1}{12} q (3L)^2$$

$$\bar{\Phi}_D^{(3)} = \frac{2EJ}{3L} \left[2\psi_D - 3 \cdot \frac{1}{3}\psi \right] + \frac{1}{12} q (3L)^2$$

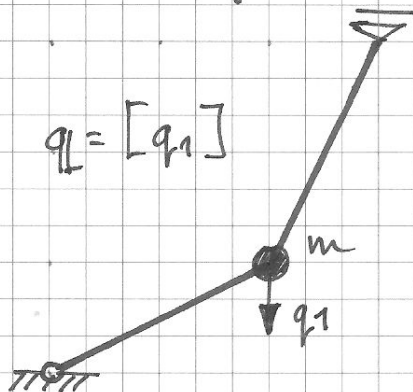
$$\psi_D = -0,702 \frac{qL^3}{EJ}$$

$$\psi = -1,858 \frac{qL^3}{EJ}$$

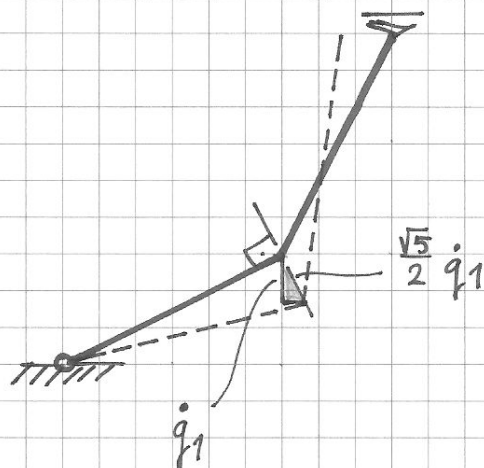
$$\bar{\Phi}_A^{(3)} = 0,0206 qL^2$$

Egzamin z MK 2, 11 IX 2014, zadanie 2

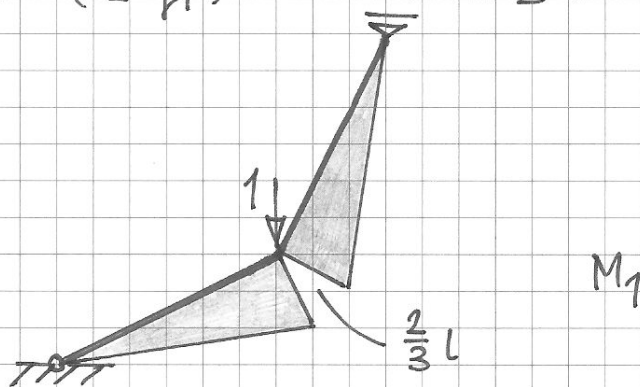
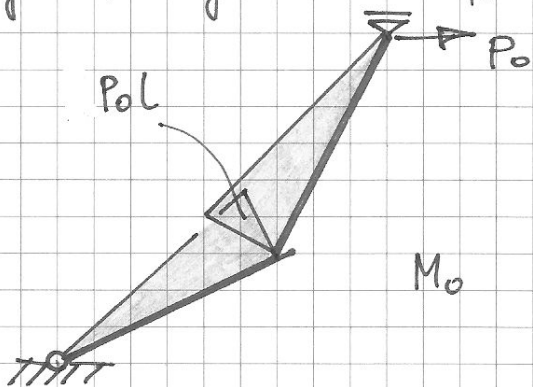
Współrzędna Lagrange'a



Plan prędkości



Energia kinetyczna: $2E_k = m \left(\frac{\sqrt{5}}{2} \dot{q}_1 \right)^2 \rightarrow M = \left[\frac{5}{4} m \right]$



$$d_{11} = 0,663 \frac{L^3}{EJ}$$

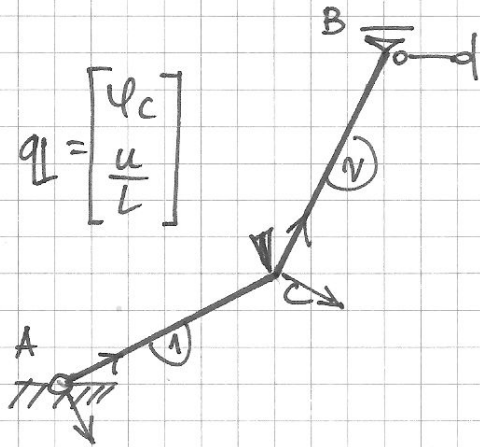
$$d_{10} = -0,994 \frac{P_0 L^3}{EJ}$$

$$(1 - \theta^2 d_{11} \cdot \frac{5}{4} m) a_1 = d_{10} \quad a_1 - \text{amplituda } q_1$$

$$a_1 = -5,784 \frac{P_0 L^3}{EJ}$$

Schemat geometryczne wyznaczamy

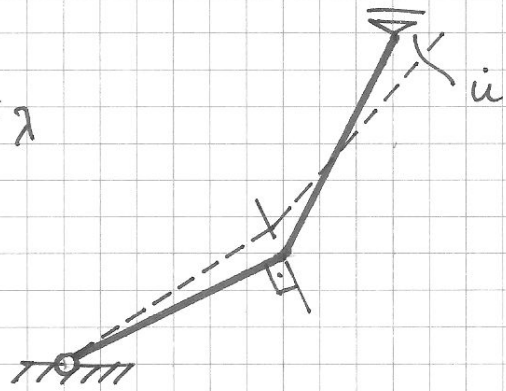
Plan prędkości



$$q = \begin{bmatrix} \varphi_c \\ \frac{u}{L} \end{bmatrix}$$

$$\lambda^{(1)} = \lambda^{(2)} = \sqrt{5} \lambda$$

$$\lambda = L \sqrt[4]{\frac{\mu \omega^2}{EJ}}$$



$$W_A^{(1)} = 0 \quad W_C^{(1)} = -\frac{\sqrt{5}}{3} u \quad u^{(1)} = 0$$

$$W_C^{(2)} = -\frac{4}{3\sqrt{5}} u \quad W_B^{(2)} = \frac{2}{\sqrt{5}} u \quad u^{(2)} = \frac{1}{\sqrt{5}} u$$

Równania równowagi:

$$\Phi_C^{(1)} + \Phi_C^{(2)} = 0$$

$$W_C^{(1)} \cdot W_C^{(1)} + W_C^{(2)} \cdot W_C^{(2)} + W_B^{(2)} \cdot W_B^{(2)} - B_{||}^{(2)} \cdot u^{(2)} = 0$$

Wzory transformacyjne i siły bezwładności:

$$\Phi_C^{(1)} = \frac{EJ}{\sqrt{5}L} \left[\alpha'(\sqrt{5}\lambda) \varphi_c + \nu'(\sqrt{5}\lambda) \frac{u}{3L} \right]$$

$$\Phi_C^{(2)} = \frac{EJ}{\sqrt{5}L} \left[\alpha'(\sqrt{5}\lambda) \varphi_c - \nu'(\sqrt{5}\lambda) \frac{4u}{15L} - \delta'(\sqrt{5}\lambda) \frac{2u}{5L} \right]$$

$$W_C^{(1)} = -\frac{EJ}{5L^2} \left[\nu'(\sqrt{5}\lambda) \varphi_c + \gamma'(\sqrt{5}\lambda) \frac{u}{3L} \right]$$

$$W_C^{(2)} = \frac{EJ}{5L^2} \left[\nu'(\sqrt{5}\lambda) \varphi_c - \gamma'(\sqrt{5}\lambda) \frac{4u}{15L} - \varepsilon'(\sqrt{5}\lambda) \frac{2u}{5L} \right]$$

$$W_B^{(2)} = -\frac{EJ}{5L^2} \left[\delta'(\sqrt{5}\lambda) \varphi_c - \varepsilon'(\sqrt{5}\lambda) \frac{4u}{15L} - \chi'(\sqrt{5}\lambda) \frac{2u}{5L} \right]$$

$$B_{||}^{(2)} = \omega^2 \cdot \mu \cdot \sqrt{5}L \cdot \frac{1}{\sqrt{5}} u = \frac{EJ}{L^2} \lambda^4 \frac{u}{L}$$