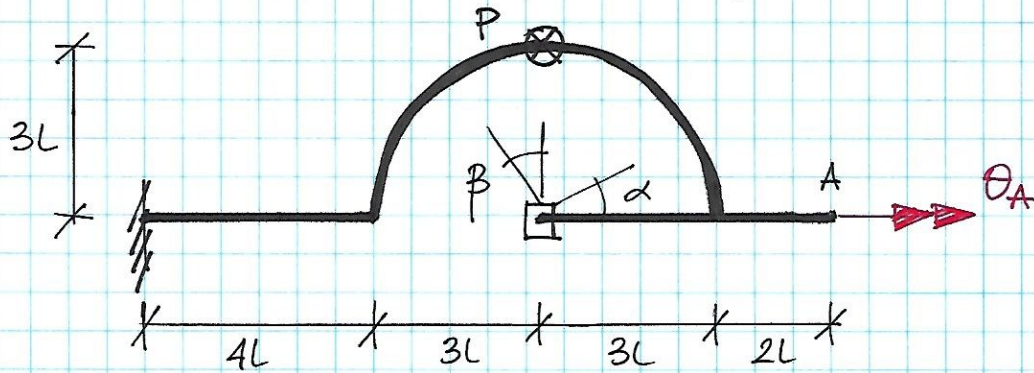


Oblicz kąt obrotu θ_A . $\frac{1}{2}EJ = GJs$

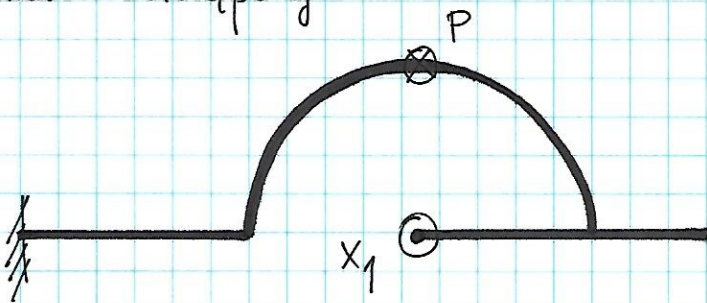


$$\alpha \in (0, \frac{1}{2}\pi)$$

$$\beta \in (0, \frac{1}{2}\pi)$$

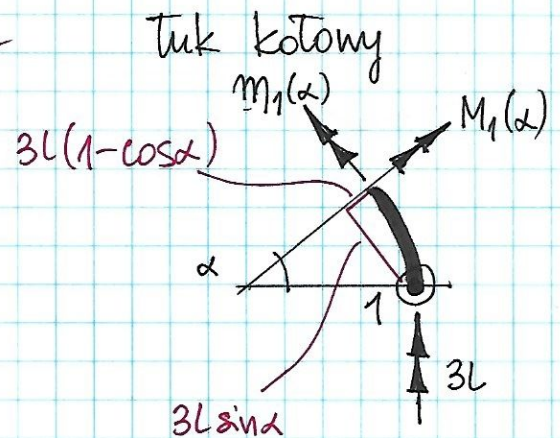
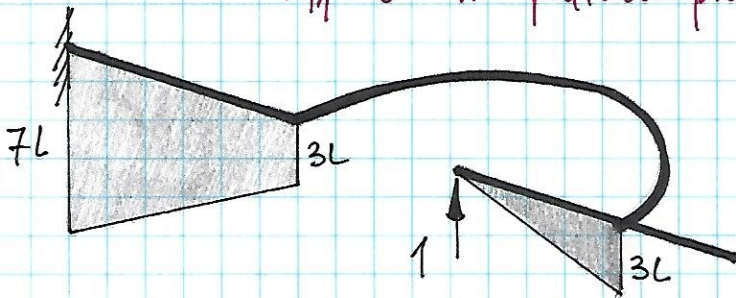
$$\beta = \alpha - \frac{1}{2}\pi$$

Schemat zastępczy



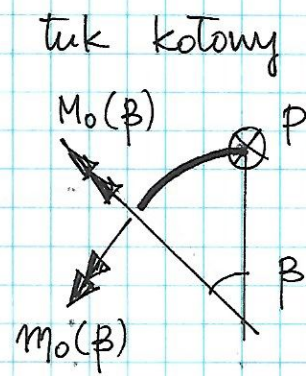
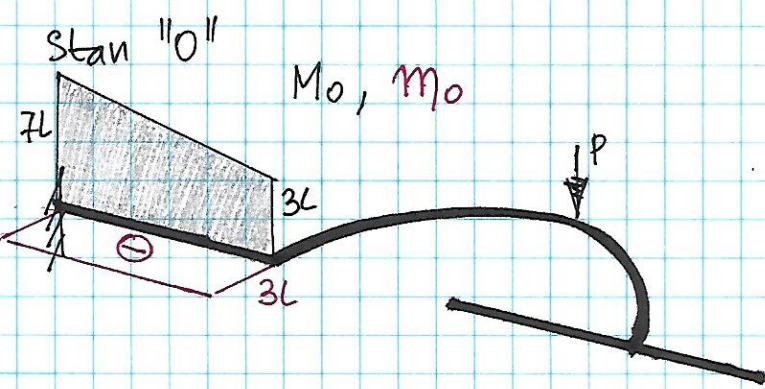
$$X_1 = 1$$

M_1
 $m_1 = 0$ na przętkach prostych



$$M_1(\alpha) = 0, \quad m_1(\alpha) = -3L$$

$$\delta_{11} = \frac{1}{EJ} [114,33 L^3] + \frac{1}{GJs} [84,82 L^3] = 283,98 \frac{L^3}{EJ}$$



$$M_o(\beta) = -3PL \cos \beta$$

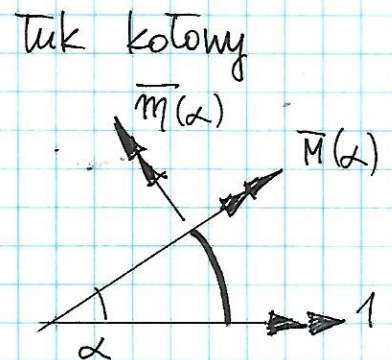
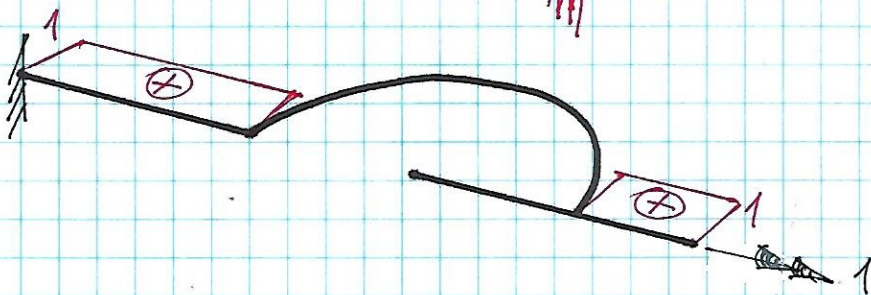
$$m_o(\beta) = 3P(1 - \cos \beta)$$

$$\delta_{10} = \frac{1}{EJ} [-105,33 PL^3] + \frac{1}{GJ_s} [-15,411 PL^3] = -136,16 \frac{PL^3}{EJ}$$

$$X_1 = 0,48 P$$

Obciążenie momentem jednostkowym w punkcie A zgodnie z założonym zwrotem kąta θ_A

$\bar{M} = 0$ na prętach prostych



$$\bar{M}(\alpha) = -\cos \alpha, \quad \bar{M}(\beta) = \bar{M}(\alpha - \frac{1}{2}\pi)$$

$$\bar{m}(\alpha) = \sin \alpha, \quad \bar{m}(\beta) = \bar{m}(\alpha - \frac{1}{2}\pi)$$

$$\theta_A = \frac{1}{EJ} \int_{\text{tuk}} (M_1 X_1 + M_o) \bar{M} ds + \frac{1}{GJ_s} \int_{\text{tuk}} (m_1 X_1 + m_o) \bar{m} ds = -27,08 \frac{PL^3}{EJ}$$

Opracował Szymon Spodzieja