

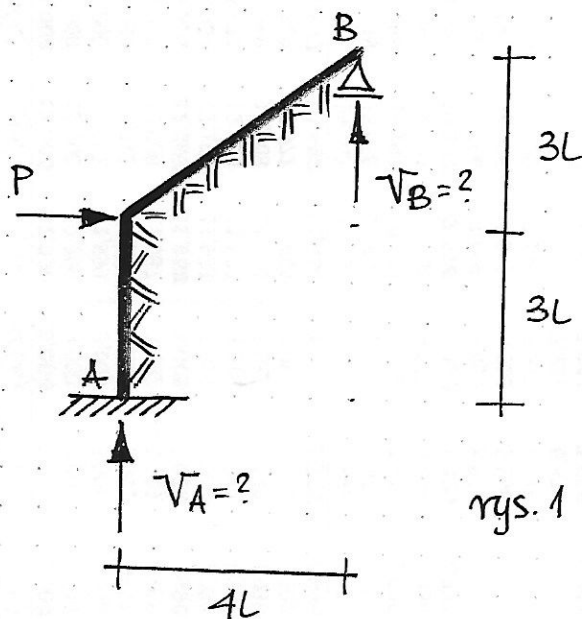
Egzamin z Mechaniki Konstrukcji (MK IPB), 24.06.2017
studia niestacjonarne

NAZWISKO, Imię				
rok akademicki zaliczenia ćwiczeń	nr albumu	grupa (IPB / BZ)	tryb studiów (ST / NST)	
ocena zadania 1	ocena zadania 2	ocena zadania 3	ocena egzaminu	ocena łączna

Zadanie 1.

$EJ = const., \quad k = 0,0064 \frac{EJ}{l^4}$

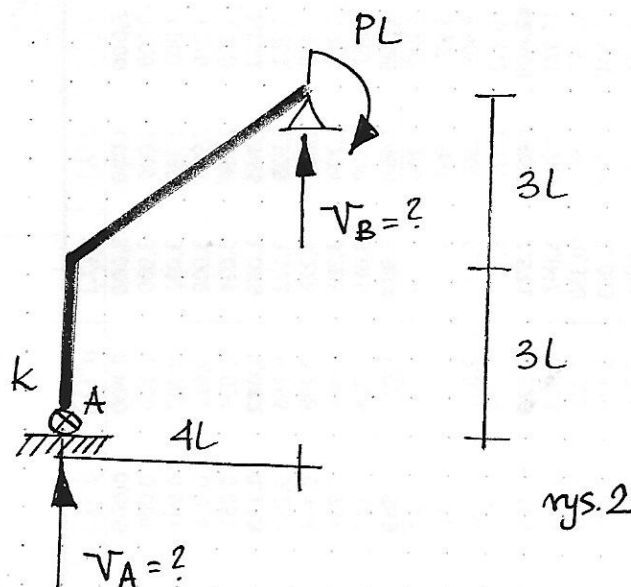
Oblicz reakcję V_A w podporze A oraz reakcję V_B w podporze B w ramie z rys. 1.



Zadanie 2.

$EJ = const., \quad k = 4 \frac{EJ}{l}$

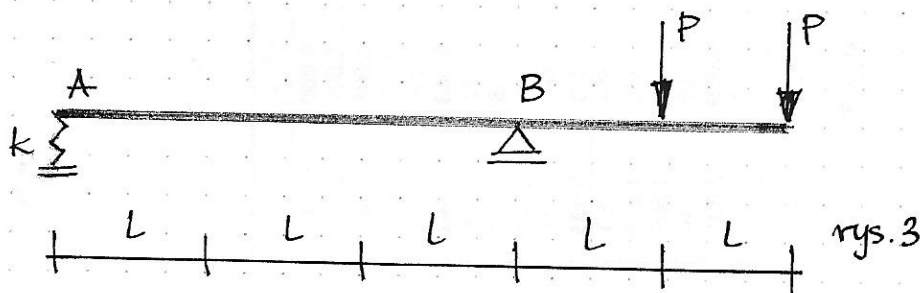
Oblicz reakcję V_A w podporze A oraz reakcję V_B w podporze B w ramie z rys. 2



Zadanie 3.

$EJ = const., \quad k = 10 \frac{EJ}{l^3}$

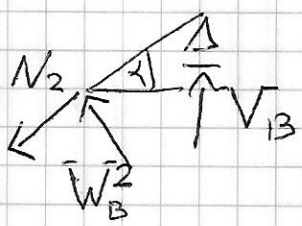
Wyznacz linię ugięcia odcinka $A-B$ belki z rys. 3.



Sily w węzle B:

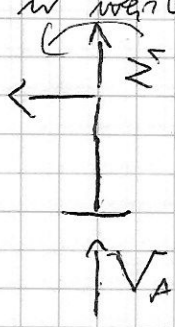
$$\sin d = \frac{3}{5}$$

$$\cos d = \frac{4}{5}$$



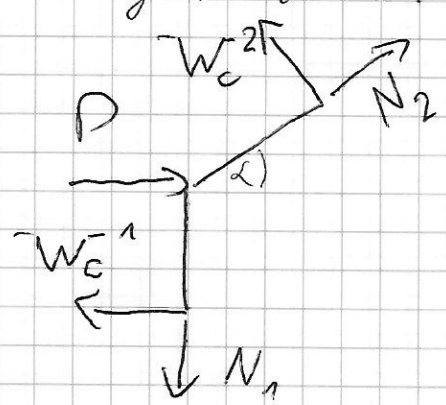
$$V_B = -\frac{W_B^2}{\cos d} = -\frac{5}{4} W_B^2$$

Sily w węzle A:



$$V_A = -N_1$$

Sily w węzle C:



$$N_1 = W_C^2 \cdot \frac{1}{\cos d} + (W_C^1 - P) \frac{\sin d}{\cos d}$$

$$V_A = -\frac{5}{4} W_C^2 - (W_C^1 - P) \frac{3}{4}$$

RR:

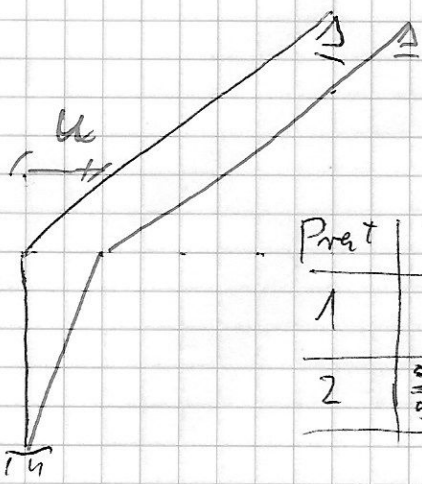
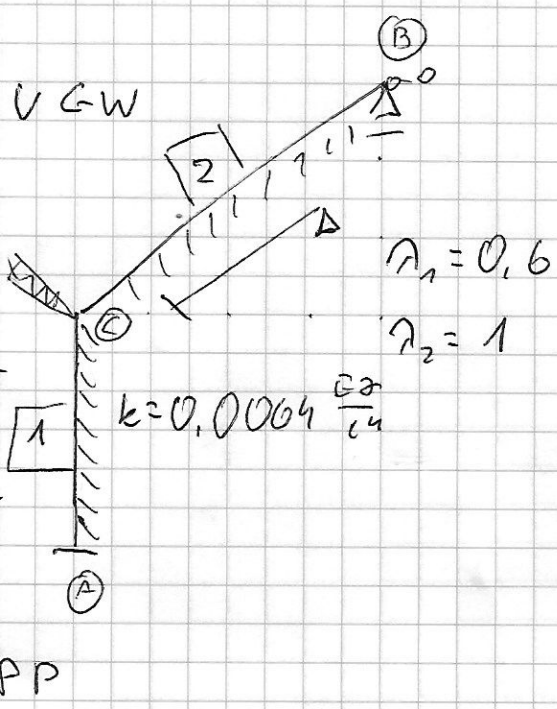
$$\Phi_C^1 + \Phi_C^2 = 0$$

$$-\left[W_C^1 \bar{u} + W_C^2 \frac{3}{5} \bar{u} + W_B^2 \frac{3}{5} \bar{u} \right] + P \bar{u} = 0$$

$$\frac{EJ}{L} \begin{bmatrix} 1,95 & -0,658 \\ -0,658 & 0,463 \end{bmatrix} \begin{bmatrix} \varphi_C \\ \frac{u}{L} \end{bmatrix} = \begin{bmatrix} 0 \\ 1 \end{bmatrix} PL$$

$$\varphi_C = 1,400 \frac{PL^2}{EJ}$$

$$\frac{u}{L} = 4,149 \frac{PL^2}{EJ}$$



Przet	*W	w*
1	0	u
2	$\frac{3}{5}u$	$\frac{3}{5}u$

$$W_B^2 = -0,130P$$

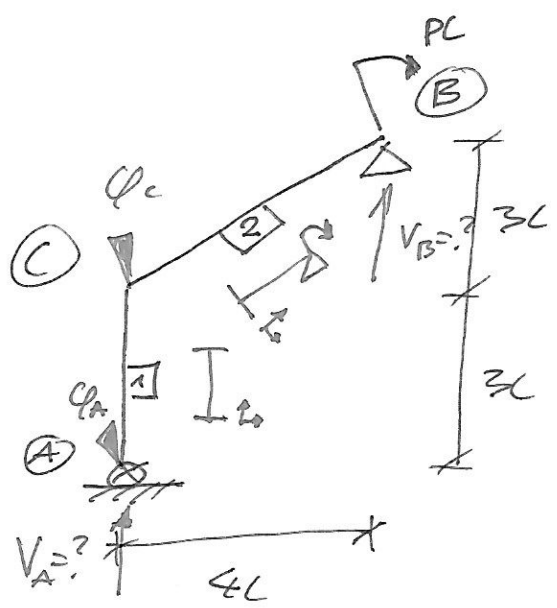
$$V_B = 0,1625P$$

$$W_C^2 = 0,236P$$

=>

$$W_C^1 = 0,936P$$

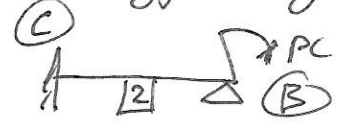
$$V_A = -0,247P$$



$k = 4 \frac{EJ}{L}$, $EJ = \text{const}$

$q_1 = \begin{bmatrix} \varphi_A \\ \varphi_C \end{bmatrix}$

Moment wyjściowy:



$\Phi_c^{(02)} = \pm \frac{PL}{2}$

R.R.M.P.:

1) $\Phi_A^{(1)} + Q_{\varphi_A} = 0$

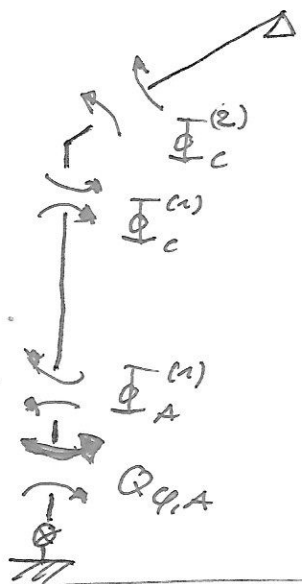
2) $\Phi_c^{(1)} + \Phi_c^{(2)} = 0$

Wzory transformacyjne:

$\Phi_A^{(1)} = \frac{2EJ}{3L} [2\varphi_A + \varphi_C - 3 \cdot 0]$

$\Phi_c^{(1)} = \frac{2EJ}{3L} [\varphi_A + 2\varphi_C - 3 \cdot 0]$

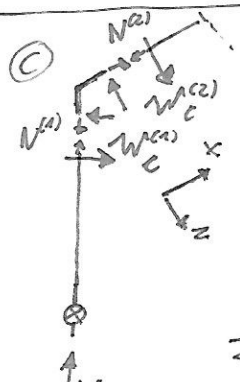
$\Phi_c^{(2)} = \frac{3EJ}{5L} [\varphi_C - 0] = \Phi_c^{(02)}$



Zwizczek konstytatywny podpory podalnej:

$\varphi_{A,P} = \varphi_A$, $\Delta\varphi_A = \varphi_{A,P} - \varphi_{A,L} = \varphi_A$
 $\varphi_{A,L} = 0$
 $Q_{\varphi_A} = k \cdot \Delta\varphi_A = \frac{EJ}{L} [4\varphi_A]$

$\Rightarrow \frac{EJ}{L} \begin{bmatrix} \frac{16}{3} & \frac{2}{3} \\ \frac{2}{3} & \frac{29}{15} \end{bmatrix} \begin{bmatrix} \varphi_A \\ \varphi_C \end{bmatrix} = \begin{bmatrix} 0 \\ -\frac{PL}{2} \end{bmatrix} \Rightarrow \varphi_A = 0.034 \frac{PL^2}{EJ}$ wz. trans $M_c^{(1)} = 0.158P$
 $\varphi_C = -0.270 \frac{PL^2}{EJ} \Rightarrow M_c^{(2)} = 0.268P$



Równanie równowagi węzła C za oś z:

$\Sigma z: \frac{4}{5} N^{(1)} - \frac{3}{5} M_c^{(1)} - M_c^{(2)} = 0$

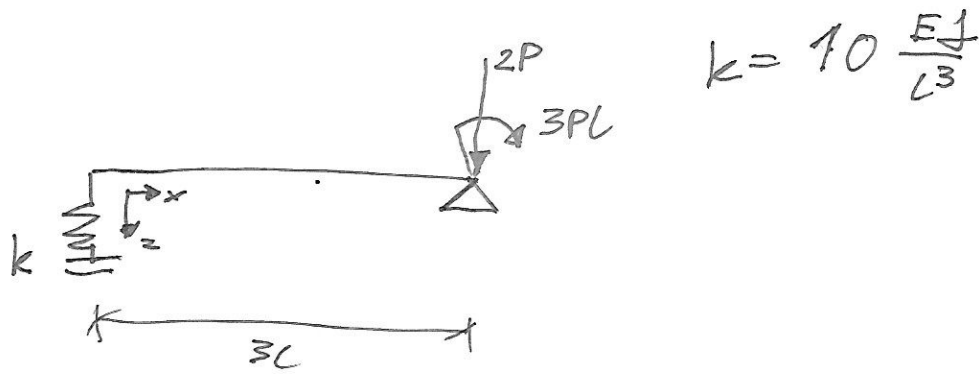
$N^{(2)} = \frac{3}{4} M_c^{(1)} + \frac{5}{4} M_c^{(2)} = 0.453P$

Z równowagi pionowego przęta na pion:

$V_A + N^{(2)} = 0 \Rightarrow V_A = -N^{(2)} = -0.453P$

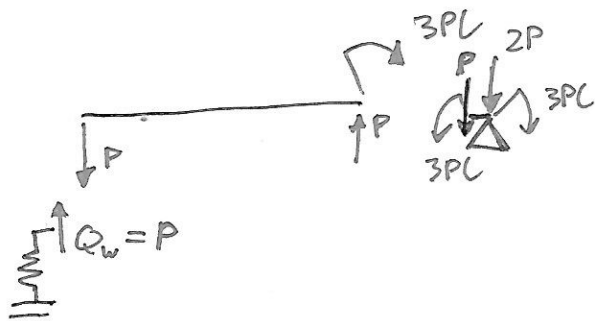
Z równowagi globalnej na pion: $V_A + V_B = 0 \Rightarrow V_B = -V_A = 0.453P$

Redukcja wspornika:



$$k = 10 \frac{EI}{L^3}$$

Belka jest statycznie wyznaczalna, więc łatwo dostajemy:



zw. konst. podjęmy pod uwagę:

$$Q_w = k \cdot \Delta w \Rightarrow \Delta w = \frac{P}{k}$$

$$\text{Mamy } \Delta w = 0 - w(0) \Rightarrow w(0) = -\frac{P}{k} = -\frac{1}{10} \frac{PL^3}{EI}$$

$$\text{Mamy: } w(x) = C_0 + C_1 x + C_2 x^2 + C_3 x^3$$

Warunki brzegowe (co najmniej 2 kinematyczne):

$$w(0) = -\frac{1}{10} \frac{PL^3}{EI} \quad w(3L) = 0$$

$$M(0) = 0 \quad M(3L) = -3PL$$

$$\Rightarrow w(x) = \frac{PL^3}{EI} \left[\frac{1}{10} - \frac{22}{5} \left(\frac{x}{3L} \right) + \frac{9}{2} \left(\frac{x}{3L} \right)^3 \right]$$